### Super Seal: Development of a Robust 2<sup>nd</sup> Stage Oil Sealing Device for Heavy Duty Engines.

Sponsor: Cummins Inc., Liaison Engineer - Terry ShawFaculty Advisor: Dr. William OatesCourse Instructor: Dr. Chiang Shih

#### Team 1 Members:

cummins

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**Final Presentation** 

April 13th, 2017

Cummins' Heavy Duty Truck engine, the ISX 15 @ 15 Liters, 600 HP

## Presentation Overview

- Project Review
  - Background Information
  - Project Description

#### Concept Design

- Test Rig Design
- Seal Design
- Testing
- Project Schedule
- Conclusion



# Project Background

### What's The Problem?

- Motor oil is repeatedly leaking past the rear crankshaft seal.
  - Failed seal
  - Material fluctuations due to thermal transients

### Motivation

- Cost
- Evolution of Customer Perceptions





Figure 1: Depiction of rear crank seal leaking oil.<sup>2</sup>



Figure 2: Cummins' newest engine, the Hedgehog @ 95 Liters, 4500 HP Cost for crank seal replacement: \$21,000.<sup>3</sup>

Presenter: Olaniyi Ogunbanwo

# Project Background

### Goal Statement

- Design a device to capture leaking oil from a rotating test crankshaft and deposit it into a reservoir so that is can be reintroduced to the crankcase.
  - Primarily demonstrate functionality/performance of additional seal design.
  - NOT to demonstrate life capabilities of design solution.

### **Objectives**

- Design a capturing device to collect oil.
- Design a rig that can be used to test the recapture device.
- Determine feasibility of each design with technical proof.
- Construct the oil recapture device and test rig.
- Perform the 24-hour trial, and assess overall project success.





# Test Rig Concept Generation



Macroscopic Ideation



# Sorting Test Cycle



Initial Test Cycle							
Duration (hrs)	Speed (RPM)						
2	500						
6	2000						
2	500						
14	0						

### Requirements

- Shell must be oil tight.
- Shell will be filled with oil to 55 mm above the bottom of the 165 mm crankshaft seal.
- Oil must be heated to 125°C.

If the sorting cycle yields positive results, additional steps and durations can be added to increase complexity.

**Presenter: Christian Milione** 



## Macroscopic Design



# Major 'OTC' Components

- 1500W Band Heater
  - 900° F at 120 Volts.
  - Dimmer Switch to alter voltage input to heat crankcase to 125° C

### 1/3 HP Belt Drive Dayton Motor

- 1475 RPM at 120 Volts.
- V-belt pulley system utilized to achieve desired shaft speeds for test.

### Mounted Bearings

- Pillow Block Bearings with 1" bore to support custom shaft.
- Air Compressor
  - Craftsman 4 gallon air compressor.



Figure 7: Pictures of components purchased for the test rig.

**Presenter: Christian Milione** 

# Major Custom Components

- Custom Test Shaft
  - Custom flanges: (1) with 165 mm and (1) with 140 mm OD and the labyrinth seal
    - 4140 Steel Mimic OEM crankshaft
      - Seals press fit over flanges.
  - Flanges press fit on shaft and welded in place

#### Crankcase

- Made from 8" (203 mm) schedule 40 steel pipe.
- Caps welded on either side for seals to be press fit inside.



Figure 9: Picture of the custom crankcase employed in the test rig.

Presenter: Kyle Brooks



# Seal Concept Generation



### Inherent Challenges

#### Sealing

- Each sealing method explored theoretically fails some customer requirement.
- No "1" solution.
  - Solution = Idea(A) + Idea(B) + ...

#### Innovation

- Use of innovative design techniques/materials.
- "Exciters" in addition to "expected"

#### Space

- Tight tolerances on spatial availability for device.
  - Keep in mind the customer's customer.

# Seal Concept Generation

Macroscopic Ideation – Seal Solution



Additional Secondary Contact Seal		Recollection Through A Vacuum		Pressure Cav	ity Behind	Centrifugal Pressure Seal						
(current one being used/size variant)												
PRO CON		PRO	CON	PRO	CON	PRO	CON					
In production	Fail due to dry sliding	Optimal Re-Capture	No vacuum source	Prevents Leakage	No PSI source	Non-Contact	No Sealing Stopped					
Easy to use	If lubed, fail w/ primary seal		Dry Sliding Seal Req'd	Favorable PSI Gradient	Dry Sliding Seal Req'd	Low Friction	Size Constraints?					
	Envelope too big		Primary seal distortion		Primary Seal distortion	+ Eff vs. Labyrinth	+ Cost					
	Copying failure				What if more oil gets out?							
	Labyrinth	Hybrid Labyrinth		Hybrid Labyrinth + Centrifugal Pressure Seal		New Primary Seal						
PRO	CON	PRO	CON	PRO	CON	PRO	CON					
Non-Contact	No Sealing Stopped	Non-Contact Running	Wear when Contact	HL Pushes Fluid IN	Space	Nano-Composites for Low Wear						
	Size Constraints?	Contact Slow/Stopped	Weep/Seep	Cntr Pushes Contamin. OUT	Oil Recapture	COE Advisor Wants Thi	S					
	+ Cost		Pushes Contaminants IN									

Figure 12: Decision making matrix to facilitate design.

Presenter: Kyle Brooks

# **Concept Selection**

#### Foundation of Design

Pressurize cavity behind rear main seal

- Aids in a solution to the root cause of the problem = **Best** place to start



# Seal Design

#### Labyrinth Seal

#### Mechanical Seal

- Designed to make the path of fluid tortuous through the implementation of numerous channels.
- Produces a seal when the shaft is rotating due to centrifugal motion.
- Non-Contact Seal Increased longevity.

#### Alterable Parameters

- Number of teeth
- Tooth Geometry
- Material of Seal

#### Constraints

- Target air consumption 2.25 L/s ≈ 1% of engine air intake
- Static TIR = 0.5 mm
- Dynamic TIR 0.35 mm



Figure 14: Labyrinth seal visualization.

**Presenter: Sean Casey** 



# Types of Labyrinth Seals

- Straight Through
  - Basic annular shape.
  - Different patterns of teeth.
  - Easier to machine.

- Stepped
  - Conical in shape.
  - More difficult to machine.
  - Not much more efficient for greater cost to produce.







## Design of Labyrinth Seal

- Interlocking Teeth
  - Optimal solution is 10 teeth.
  - Ideal for Cummins.
  - Difficult to machine and assemble.

### Single Row of Teeth

- Optimal solution is 15 teeth.
- Ideal for this scenario.
- Easier to machine and assemble.



% Consumption with Increased Seal # of Teeth Interlocking Configuration

Presenter: Sean Casey

# **Our Super Seal**

#### 15 tooth system

Created by stacking sheet metal circles \_ of alternating diameters.



**Presenter: Sean Casey** 

Figure 19: CAD vs. assembled labyrinth seal.

# **Project Economics**

#### **Bill of Materials**

Part Description	]	Total Price	Quantity		Cost
motor	\$	147.24	1	\$	147.24
3ft drive shaft	\$	16.62	1	\$	16.62
Rectangular Tube 6"x2" 2 ft long steel for mounting brac	\$	47.38	1	\$	47.38
Pulley 1: 3" diameter 1/2" bore	\$	30.00	1	\$	30.00
Pulley 2: 3.5" diameter 1" bore	\$	30.00	1	\$	30.00
Pulley3: 5" diameter 1/2" bore	\$	30.00	1	\$	30.00
Pulley 4: 9" diameter 1" bore	\$	30.00	1	\$	30.00
Belt1: 4LX50" V-belt	\$	12.00	1	\$	12.00
Belt2: 4LX56" V-belt	\$	12.00	1	\$	12.00
tubing (pressure lines) 1/4" ID 1/2" OD High Temp silico	\$	32.55	1	\$	32.55
Bearings (mounted) steel bearings	\$	39.72	2	\$	79.44
Washers	\$	1.05	6	\$	6.30
bolts M10 (pack)	\$	10.45	1	\$	10.45
nuts M10	\$	0.48	30	\$	14.40
Base Plate	\$	59.00	1	\$	59.00
heating element	\$	102.00	1	\$	102.00
rear crankshaft seal	\$	-	1	\$	
front crankshaft seal	\$	-	1	\$	
pressure regulator/ safety relief valve	\$	14.99	2	\$	29.98
primary enclosure (crankcase)	\$	75.67	1	\$	75.67
Secondary enclosure (super seal)			1	\$	
terciary enclosure	\$	35.00	1	\$	35.00
labyrinth seal	\$	20.00	1	\$	20.00
T's for pressure reg and dial	\$	7.93	2	\$	15.86
Plexi glass shield for system 48" x 96" x 1/8" Clear Acryli	\$	99.00	1	\$	99.00
Oil (1 quart?)	\$	10.00	1	\$	10.00
Containers	\$	2.57	2	\$	5.14
Plug for motor 6" 6 gage wire, max amp: 50 125/250 VAC	\$	9.56	2	\$	19.12
Dimmer Switch for heater 1500 watt	\$	63.22	1	\$	63.22
barbed pipe fittings for tee and pressure gauge	\$	3.85	2	\$	7.70
Caps for crankcase ends	\$	132.52	1	\$	132.52
flanges for shaft seals inside dia fit			1	\$	
flange for crankcase			1	\$	
2 end threaded pipe nipple for pressure Tee	\$	4.59	2	\$	9.18
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**Presenter: Jonathan Strickland** 

curumins.



Expenditure Breakdown Relative to Budget



## **Project Economics**



# Project Schedule (Gantt Chart)



				2016			2017				
Name	2	Begin date	End date	September	October	November	December	January	l February	March	April
0	General Research	9/12/16	9/26/16								
0	Needs Assessment	9/19/16	9/26/16								
0	Conceptual Design	9/20/16	11/1/16		0 0 0 0 0						
0	CAD Renderings	10/24/16	11/21/16								
0	Finalize concepts	11/21/16	12/20/16								
0	Finalize BOM	1/9/17	2/15/17								
0	Order Parts	1/16/17	3/15/17						2000		
0	Air flow/Seal DFM	2/15/17	3/1/17							1	
0	Assemble Test Rig	2/20/17	3/30/17							0000	
0	Performance Trials Test	3/15/17	3/31/17								1
0	Test Sealing Solution	4/3/17	4/10/17								
0	Analyze results/ Adjustm	.4/10/17	4/13/17								

### Future Recommendations

### If this project has successors:

- Redesign and enhance safety enclosure.
- Investigate alternative materials for the labyrinth seal.
- Plan and conduct a more rigorous trial.
- Develop modular sealing alternatives for comparison.
- Try to decrease the overall complexity of test rig assembly.
- Examine changes that can be made to the OEM seal to address the root cause.





# Conclusion Project Goal



- Design a device to capture leaking oil from a rotating test crankshaft and deposit it into a reservoir so that is can be reintroduced to the crankcase.
  - Paying close attention to the test rig.

### Ideal Design

- Design for test rig and capture device is finalized.
  - Minor changes may still be implemented.
  - Paying close attention to tolerances.

### What's Next?

Testing

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**Presenter: Jonathan Strickland** 





# Questions?